

Second order systems

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Trial lecture at the Department of Electrical Engineering and Computer Science,
University of Stavanger, Norway, 2017

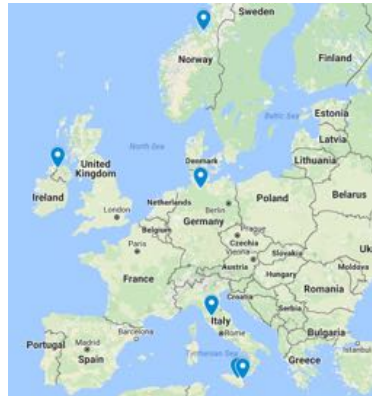
About Me

Education:

- PhD in Engineering Cybernetics, Norwegian University of Science and Technology (NTNU), Norway
- MSc in Computer Science Engineering, University of Siena, Italy
- BSc degree in Computer Science Engineering, University of Catania, Italy

Mobility:

- Visiting Fellow, Technical Aspects of Multimodal Systems (TAMS), Department of Mathematics, Informatics and Natural Sciences, University of Hamburg, Hamburg, Germany
- Visiting Student, School of Computing and Intelligent Systems, University of Ulster, Londonderry, United Kingdom
- Granted with an Erasmus+ Staff Mobility for Teaching and Training project



Activities:

- Membership Development Officer for the IEEE Norway Section

Research topics

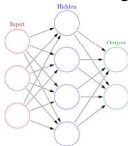
Visualisation/Game Development



Mobile Device



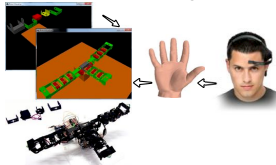
Artificial Intelligence



Augmented Reality/Virtual Reality



Software/Hardware
Codesign



Safety-Critical Systems



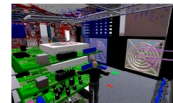
Micro-controllers, IoT, Maker Tech.



Real-time Systems



Education



About Me

Current position:

- Filippo Sanfilippo, Postdoctoral Fellow at the Dept. of Eng. Cybernetics, NTNU, Trondheim, Norway

Current courses:

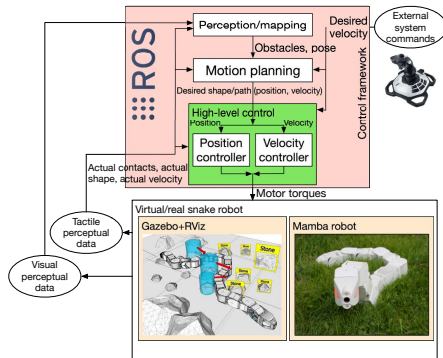
- TTK4235 - Embedded Systems (Lecturer)
- Experts in Teamwork - Snake robots (Supervisor)

Past courses:

- Real-time Computer Programming (Lecturer)
- Mechatronics, Robots and Deck Machines (Teaching Assistant)
- System Simulation in Matlab/Simulink (Lecturer)

Current research topic:

- "SNAKE - Control Strategies for Snake Robot Locomotion in Challenging Outdoor Environments", project number 240072, supported by the Research Council of Norway through the *Young research talents funding scheme*



First-order linear systems

Canonical homogeneous first-order differential equation:

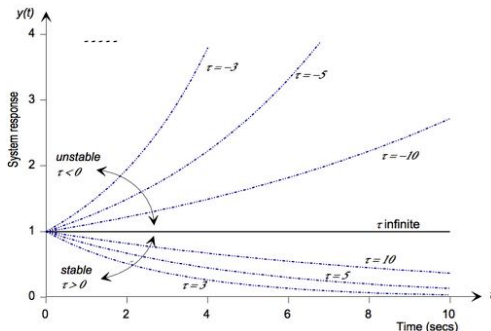
$$\tau \frac{dy(t)}{dt} + y(t) = f(t), \quad (1)$$

where $\tau \neq 0$ is the system *time constant*. The characteristic equation is given by:

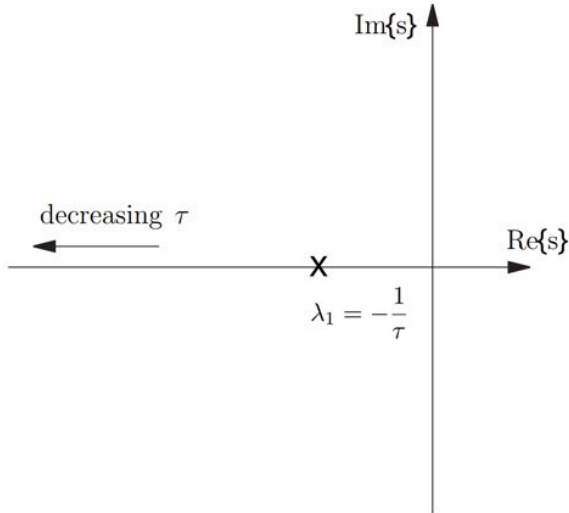
$$\tau \lambda + 1 = 0, \quad (2)$$

which has a single root, $\lambda = -1/\tau$. The system response to an initial condition $y(0)$ is:

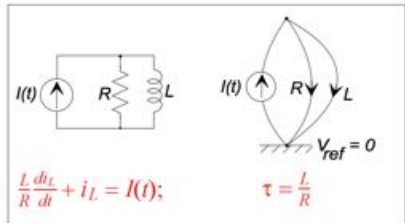
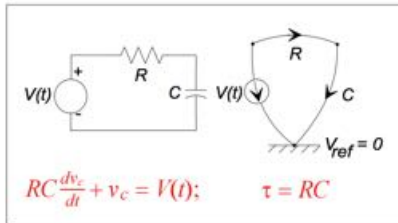
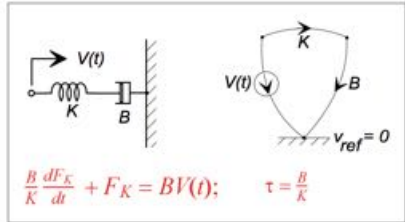
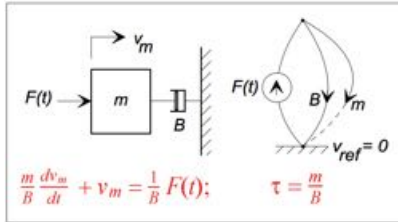
$$y_h(t) = y(0)e^{\lambda t} = y(0)e^{-t/\tau}, \quad (3)$$



First-order linear systems: s-plane



First-order linear systems



The characteristic response of first-order systems

- The canonical homogeneous first-order differential equation is given by:

$$\tau \frac{dy(t)}{dt} + y(t) = f(t). \quad (4)$$

The solution (given $f(t)$ and $y(0) = 0$) is the *characteristic first-order response*.

- The first-order homogeneous solution is of the form of an exponential function:

$$y_h(t) = e^{-\lambda t}, \lambda = 1/\tau. \quad (5)$$

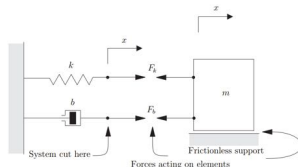
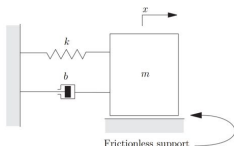
- The total response $y(t)$ is the sum of two components:

$$y(t) = y_h(t) + y_p(t) = ce^{-t/\tau} + y_p(t), \quad (6)$$

where c is a constant to be found from the initial condition $y(0) = 0$, and $y_p(t)$ is a particular solution for the given forcing function $f(t)$.

Input $u(t)$	Characteristic Response	Input/Output Response $y(t)$ for $t \geq 0$
$u(t) = 0$		$y(t) = y(0)e^{-t/\tau}$
$u(t) = u_r(t)$	$y_r(t) = t - \tau (1 - e^{-t/\tau})$	$y(t) = [q_0 t + (q_1 - q_0 \tau) (1 - e^{-t/\tau})]$
$u(t) = u_s(t)$	$y_s(t) = y_s(t) = 1 - e^{-t/\tau}$	$y(t) = [q_0 - (q_0 - \frac{q_1}{\tau}) e^{-t/\tau}]$
$u(t) = \delta(t)$	$y_\delta(t) = \frac{1}{\tau} e^{-t/\tau}$	$y(t) = \frac{q_1}{\tau} \delta(t) + (\frac{q_0}{\tau} - \frac{q_1}{\tau^2}) e^{-t/\tau}$

Second-order linear systems



$$-F_b - F_k = -b \frac{dx}{dt} - kx = m \frac{d^2x}{dt^2} \Rightarrow m \frac{d^2x}{dt^2} + b \frac{dx}{dt} + kx = 0. \quad (7)$$

- $x(0) = x_0$, $v(0) = \dot{x}(0) = v_0$ are required initial conditions
- assume that $x(t)$ takes the form $x(t) = ce^{st}$

$$ms^2 ce^{st} + bsce^{st} + kce^{st} = 0 \Rightarrow ms^2 + bs + k = 0. \quad (8)$$

$$s_1 = -\frac{b}{2m} + \frac{\sqrt{b^2 - 4mk}}{2m}, s_2 = -\frac{b}{2m} - \frac{\sqrt{b^2 - 4mk}}{2m}. \quad (9)$$

- s_1, s_2 are the pole locations (natural frequencies) of the system.
- In most cases ($b^2 \neq 4mk$) and the initial condition response will take the form:

$$x(t) = c_1 e^{s_1 t} + c_2 e^{s_2 t}. \quad (10)$$

[1] _____

[1] MIT OpenCourseWare. Review of complex numbers. 2017. URL: <https://goo.gl/utPL5y>.

Standard terms

The pole locations are parameterized in terms of the natural frequency ω_n , and the damping ratio ζ where:

$$\omega_n = \sqrt{\frac{k}{m}}, \quad (11)$$

$$\zeta = \frac{b}{2\sqrt{km}}. \quad (12)$$

Natural frequency and damping ratio:

- The natural frequency, ω_n , is the frequency at which the system would oscillate if the damping, b , were zero
- The damping ratio, ζ , is the ratio of the actual damping, b , to the critical damping, $b_c = 2\sqrt{km}$

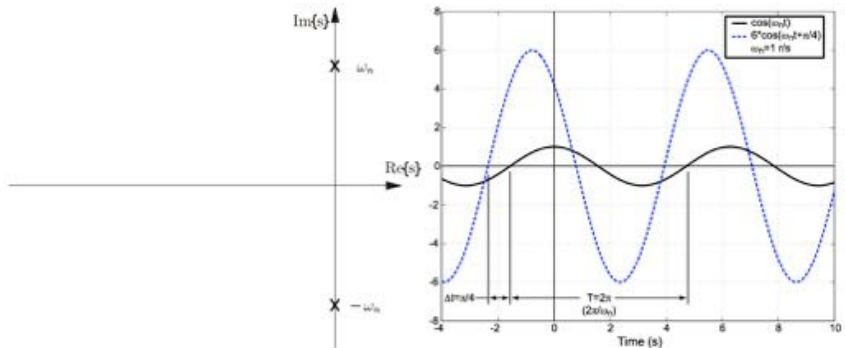
$$m \frac{d^2x}{dt^2} + b \frac{dx}{dt} + kx = 0 \Rightarrow \frac{1}{\omega_n^2} \frac{d^2x}{dt^2} + \frac{2\zeta}{\omega_n} \frac{dx}{dt} + x = 0. \quad (13)$$

Physically reasonable assumption: the values of m , and k are greater than zero (to maintain system order) and that b is non-negative (to keep things stable).

Pole locations

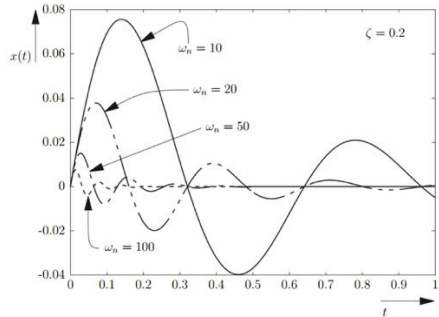
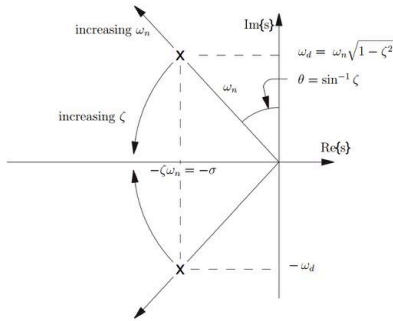
- If $b = 0$, the poles are complex conjugates on the imaginary axis at $s_1 = +j\sqrt{k/m}$, $s_2 = -j\sqrt{k/m}$. This corresponds to $\zeta = 0$, and is referred to as the undamped case
- If $b^2 - 4mk < 0$ then the poles are complex conjugates lying in the left half of the s-plane. This corresponds to the range $0 < \zeta < 1$, and is referred to as the underdamped case
- If $b^2 - 4mk = 0$ then the poles coincide on the real axis at $s_1 = s_2 = -b/2m$. This corresponds to $\zeta = 1$, and is referred to as the critically damped case
- If $b^2 - 4mk > 0$ then the poles are at distinct locations on the real axis in the left half of the s-plane. This corresponds to $\zeta > 1$, and is referred to as the overdamped case

Undamped case ($\zeta = 0$)



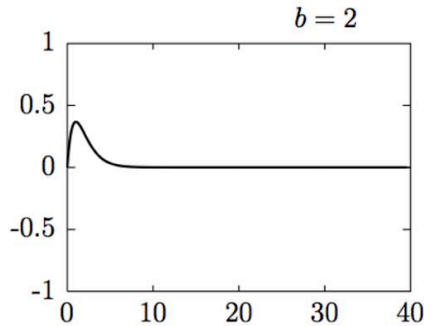
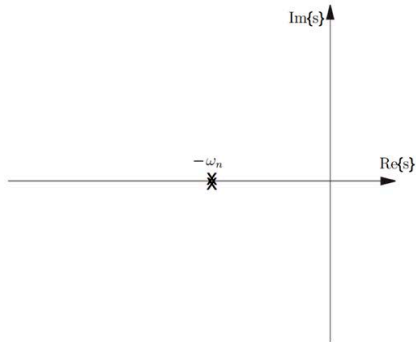
The poles lie at $s_1 = j\omega_n$ and $s_2 = -j\omega_n$.

Underdamped case ($0 < \zeta < 1$)



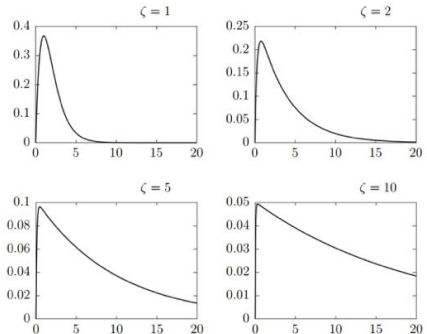
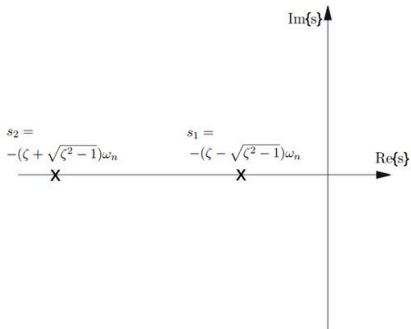
The poles lie at $s = -\sigma \pm j\omega_d$, where $\sigma = \zeta\omega_n$ is the attenuation, and $\omega_d = \omega_n\sqrt{1-\zeta^2}$.

Critically-damped case ($\zeta = 1$)



The poles lie at $s_1 = s_2 = -\omega_n$.

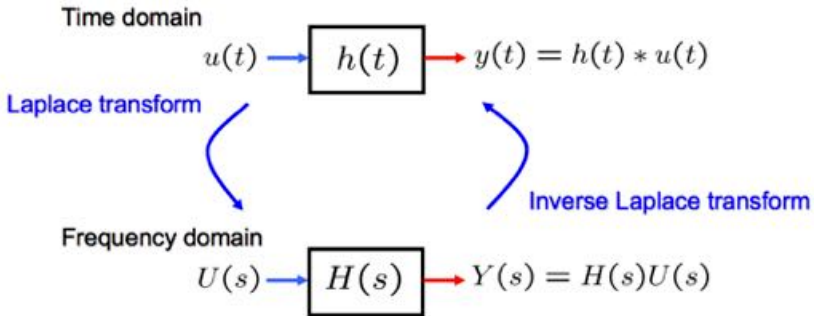
Overdamped case ($\zeta = 1$)



The poles lie at $s_1 = -(\zeta - \sqrt{\zeta^2 - 1})\omega_n$ and $s_2 = -(\zeta + \sqrt{\zeta^2 - 1})\omega_n$.

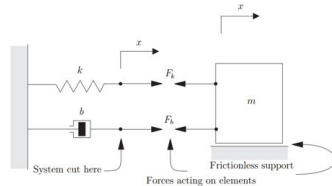
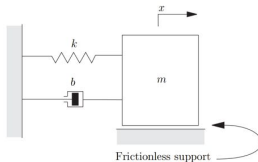
Laplace transforms and transfer functions

Laplace transforms: method for solving differential equations, converts differential equations in time t into algebraic equations in complex variable s .



- Defined as the ratio of the Laplace transform of the output signal to that of the input signal (think of it as a gain factor!)
- Contains information about dynamics of a Linear Time Invariant system

Mass-spring-damper system



ODE

$$M\ddot{y}(t) + b\dot{y}(t) + ky(t) = u(t)$$

Assume all initial conditions are zero. Then take Laplace transform,

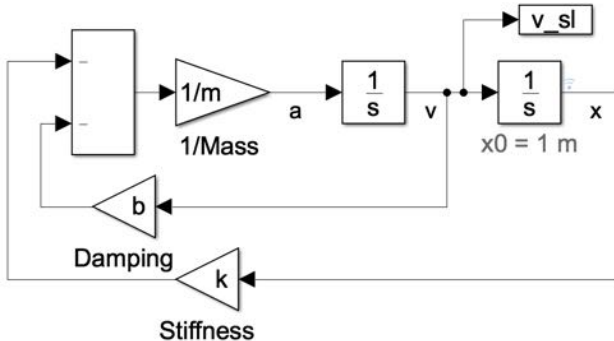
$$Ms^2Y(s) + bsY(s) + kY(s) = U(s)$$

Output \rightarrow $\boxed{\frac{Y(s)}{U(s)}} = \boxed{\frac{1}{Ms^2 + bs + k}}$ \leftarrow Transfer function

Input \rightarrow

Mass-Spring-Damper in Simulink

Simulink Model

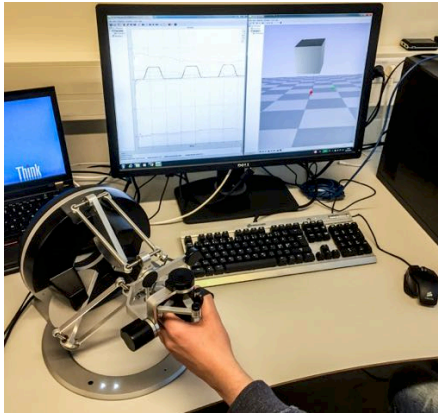


$$m \frac{d^2x}{dt} + b \frac{dx}{dt} + kx = 0. \quad (14)$$

[2] _____

[2] MathWorks. *Mass-Spring-Damper in Simulink and Simscape*. 2017. URL: <https://goo.gl/Ftr6hK>.

Applications



[3]

[3] Filippo Sanfilippo, Paul B.T. Weustink, and Kristin Ytterstad Pettersen. "A coupling library for the force dimension haptic devices and the 20-sim modelling and simulation environment". In: *Proc. of the 41st Annual Conference of the IEEE Industrial Electronics Society (IECON)*, Yokohama, Japan. 2015, pp. 168–173.

Conclusion

Learning experience:

- First-order linear systems and characteristic response
- Second-order linear systems and characteristic response
- Several applications



Thank you for your attention



Contact:

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[4–6]

- [4] Filippo Sanfilippo et al. "Virtual functional segmentation of snake robots for perception-driven obstacle-aided locomotion". In: *Proc. of the IEEE Conference on Robotics and Biomimetics (ROBIO), Qingdao, China. 2016*, pp. 1845–1851.
- [5] Filippo Sanfilippo et al. "A review on perception-driven obstacle-aided locomotion for snake robots". In: *Proc. of the 14th International Conference on Control, Automation, Robotics and Vision (ICARCV), Phuket, Thailand. 2016*, pp. 1–7.
- [6] Filippo Sanfilippo et al. "Perception-driven obstacle-aided locomotion for snake robots: the state of the art, challenges and possibilities". In: *Applied Sciences 7.4 (2017)*, p. 336.

References

- [1] MIT OpenCourseWare. *Review of complex numbers*. 2017. URL: <https://goo.gl/utPL5y>.
- [2] MathWorks. *Mass-Spring-Damper in Simulink and Simscape*. 2017. URL: <https://goo.gl/Ftr6hK>.
- [3] Filippo Sanfilippo, Paul B.T. Weustink, and Kristin Ytterstad Pettersen. "A coupling library for the force dimension haptic devices and the 20-sim modelling and simulation environment". In: *Proc. of the 41st Annual Conference of the IEEE Industrial Electronics Society (IECON), Yokohama, Japan*. 2015, pp. 168–173.
- [4] Filippo Sanfilippo et al. "Virtual functional segmentation of snake robots for perception-driven obstacle-aided locomotion". In: *Proc. of the IEEE Conference on Robotics and Biomimetics (ROBIO), Qingdao, China*. 2016, pp. 1845–1851.
- [5] Filippo Sanfilippo et al. "A review on perception-driven obstacle-aided locomotion for snake robots". In: *Proc. of the 14th International Conference on Control, Automation, Robotics and Vision (ICARCV), Phuket, Thailand*. 2016, pp. 1–7.
- [6] Filippo Sanfilippo et al. "Perception-driven obstacle-aided locomotion for snake robots: the state of the art, challenges and possibilities". In: *Applied Sciences* 7.4 (2017), p. 336.